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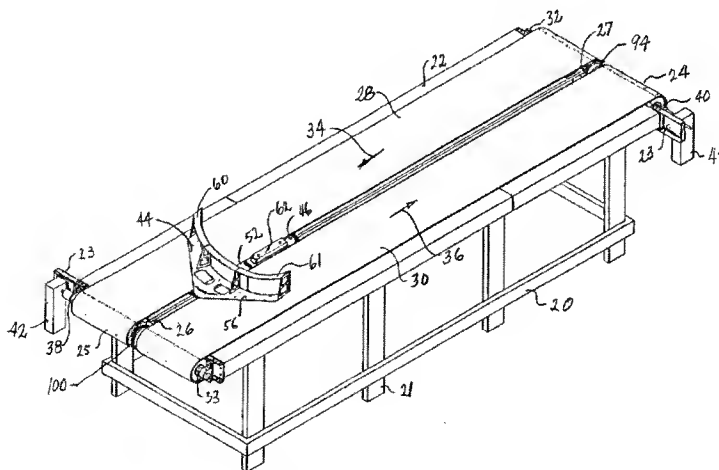
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[Continued on next page]

(54) Title: APPARATUS FOR DIVERTING A STREAM OF ARTICLES



(57) Abstract: Apparatus for diverting a stream of articles from one conveyor to another parallel conveyor moving in an opposite direction in a conveyor bed. A diverter element (44) is attached to a carrier that is driven along a path parallel to the two conveyors by a differential drive mechanism coupled to the drive mechanisms of the two conveyors. The differential drive mechanism responds to the difference in the speeds of the two conveyor drive mechanisms to drive the carrier and position the diverter element along the parallel conveyors. The diverter element (44) extends at least partway across the widths of the two conveyors to guide articles on one conveyor to the other. In a preferred version, the carrier is an endless conveyor belt (46) to which the diverter element is attached, and the differential drive mechanism includes a shaft arranged coaxially with drive or idler sprocket shafts of the conveyor belts at each end of the bed. The apparatus is useful in accumulating articles transported on a main conveyor in a first in, first-out fashion.

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APPARATUS FOR DIVERTING A STREAM OF ARTICLES**Background**

This invention relates generally to conveying systems having plural power-driven conveying sections and, more particularly, to apparatus for varying the effective length of the system by controlling the diversion of conveyed articles from one conveyor section to another.

Many conveyor applications require that articles transported on one conveyor belt be transferred to another conveyor belt that may be traveling in another direction, such as the opposite direction. Stationary rails spanning the conveying surfaces of the belts at a transfer point are often used for this purpose. In some applications, such as article accumulation, the transfer point can be moved. The moving transfer point allows the effective length of the conveying path and, consequently, the amount of articles accumulated to be controlled. In a spiral accumulator, such as that shown in US Patent No. 6,152,291, two parallel conveyor belts – an infeed belt and outfeed belt – are helically arranged with a rotatable transfer mechanism traveling between them. The position of the transfer mechanism depends on the relative speeds of the two oppositely moving belts. A guide plate on the transfer mechanism directs articles from the infeed conveyor belt to the outfeed belt. A rotatable member in the transfer mechanism simultaneously engages drivers formed on the specially designed infeed and outfeed belts. The rotatable member, driven by the drivers, translates the transfer mechanism and its attached guide plate along the belts as determined by their relative speeds. The rotatable member rides along with the transfer mechanism.

A more common accumulation system is known as a bi-di (bi-directional) table. Typically, a bi-di table includes a bi-directional conveyor belt arranged to run perpendicular to a main conveyor belt. As articles build up on the main conveyor belt, backpressure directs them onto the bi-di belt, which runs in a direction away from the main conveyor. In this way, articles are accumulated on the bi-di belt's surface. As downstream processes require more articles, the direction of the bi-di belt is reversed so that the accumulated articles are pushed back onto the main conveyor belt. These bi-di tables, which support the bi-di belt and its drive mechanisms, are generally rectangular in shape with drive or idler shafts for sprockets or pulleys at each end. But, unlike the spiral accumulator, these simple, ubiquitous tables are not first in, first out.

Thus, there is a particular need for a simple first in, first-out accumulation system, especially one that can be easily retrofitted in a bi-di table. More generally, there is also a need for a simple mechanism for diverting articles from one conveyor belt to another, such as oppositely-moving conveyor belts.

Summary

These needs and others are satisfied by a novel article-diverting apparatus embodying features of the invention. In the apparatus, a first conveyor belt is driven in a first direction at a first speed. A second conveyor belt is driven at a second speed in a different second direction, typically the opposite direction. A stream of articles is conveyed along a conveying surface of the first conveyor. A carrier travels along a path generally parallel to the first and second conveyor belts. A diverter element affixed to the carrier extends across at least a portion of the conveying surfaces of the first and second belts to divert articles in the stream from the first belt to the second belt. The carrier is controlled by a differential drive mechanism at a speed that depends on the speeds of the first and second conveyor belts. In this way, the position of the diverter element and, hence, the transfer point are determined by the relative speeds of the two belts.

In one version of the apparatus, the differential drive mechanism is stationarily attached to a framework supporting the carrier and the first and second belts. In another version, the differential drive mechanism is attached, not directly to the first and second belts, but to a first drive mechanism driving the first belt and to a second drive mechanism driving the second belt. This permits non-custom, less-expensive, standard belts to be used as the first and second belts.

In a first-in, first-out accumulator using the apparatus of the invention, a stream of articles at an upstream position on a main conveyor is supplied to the first, or infeed, conveyor belt, transferred to the second, or outfeed, conveyor belt by the diverter element on a carrier, and then fed by the second belt back to the main conveyor line at a downstream location. The amount of accumulation is determined by the carrier drive mechanism, which positions the carrier and diverter element along the infeed and outfeed belts depending on their relative speeds.

In other versions, the carrier is an endless carrier belt parallel to the first and second belts. The belt can be an intermediate belt between and abutting the first and second belts with a transfer surface portion across which products are transferred from the first belt to the second belt. Alternatively, the carrier belt flanks the first conveyor belt and carries the affixed diverter element. The flanking belt can be used in conjunction with another flanking belt on the opposite side of the second conveyor that provides two-sided support for the diverter element. In this version, the first and second conveyor belts can be abutted for direct transfer of articles from one to the other. In one version in which a carrier belt is used, the first and second belts

engage rotating elements including aligned shafts with sprockets about which the belts articulate. The differential drive mechanism is rotationally coupled to the rotating assemblies and engages the carrier belt to drive at a speed that depends on the relative speeds of the conveyor belts. This version is especially easy to retrofit in an existing bi-di table to achieve first-in, first-out capability.

In one version, the differential drive mechanism is differentially geared to the first and second drive mechanisms. But the belts and carrier can alternatively be controlled by a controller that individually controls the drive mechanisms of the first and second conveyor belts and the carrier. In either of these examples, the drive characteristics can be tailored for various applications by setting the speed of the carrier to be as_1-bs_2 , where s_1 is the speed of the first conveyor belt, s_2 is the speed of the second conveyor belt, and a and b are adjustable parameters that are chosen to fit the application.

Thus, the apparatus provides clear advantages in diverting articles from one conveyor belt to a parallel, oppositely moving conveyor belt.

Brief Description of the Drawings

These and other features, aspects, and advantages of the inventions are further addressed in the following description, appended claims, and accompanying drawings, in which:

FIG. 1 is an isometric view, partly cut away, of an apparatus embodying features of the invention, the apparatus including a diverter element for diverting a stream of articles;

FIG. 2 is a top plan view of the apparatus of FIG. 1;

FIG. 3 is an end elevation schematic of a stepped arrangement of conveying surfaces usable in the apparatus of FIG. 1;

FIG. 4 is an isometric view of a drive mechanism, including a differential drive, at one end of the apparatus of FIG. 1;

FIG. 5 is an isometric view of the opened-up differential drive mechanism of FIG. 4;

FIG. 6 is an isometric view of a drive mechanism at the opposite end of FIG. 1 from the drive mechanism in FIG. 4;

FIG. 7 is a top plan view of another version of apparatus for diverting articles, in which the diverter element is driven from one side;

FIG. 8 is a top plan view of another variation of diverter apparatus as in FIG. 7 with the diverter attached at flanking positions;

FIG. 9 is an isometric view of a portion of another version of article-diverting apparatus partly cut away to show a differential drive for a lead screw;

FIG. 10 is a side view of the lead screw drive of FIG. 9; and

FIG. 11 is a top plan schematic of a first-in, first-out accumulator application using the apparatus of FIG. 1.

Detailed Description

An apparatus embodying features of the invention for diverting articles from a stream of articles is shown in FIGS. 1 and 2. The apparatus includes a conveyor bed formed by a framework 20 with legs 21 and a pair of parallel upper support rails 22. Crossbeams 23 span the support rails at opposite ends 24, 25 of the conveyor bed. Cross supports 26 forming part of the framework support a conveyor pan or wearstrips 27 that underlie the carryway of a first infeed conveyor 28 and a parallel second outfeed conveyor 30.

Each conveyor includes an endless belt, or chain, wrapped around rotating assemblies 32, 33 at the first and second ends 24, 25 of the conveyor bed. The belt, or chain, is preferably an endless modular plastic conveyor belt, such as any of those standard belts manufactured and sold by Intralox, Inc. of Harahan, Louisiana, USA. Modular conveyor belts are preferable because the modules can be easily linked together end to end and side by side with hinge pins typically in a bricklay pattern to construct belts of almost any length and width. But any belt, including, for example, metal chains or fabric belts, could be used effectively. The infeed belt 28 and the outfeed belt 30 are driven in the direction of arrows 34, 36.

In the example of FIGS. 1 and 2, the infeed belt 28 is driven by a first infeed drive mechanism 38, which is part of the second rotating assemblies 33. The outfeed belt 30 is driven by a second outfeed drive mechanism 40, which is part of the first rotating assemblies 32. In this drive arrangement, the first and second drive mechanisms pull the belts in tension along the conveyor carryway in the direction of the arrows 34, 36. Motors 42, preferably variable-speed motors, in the infeed and outfeed drive mechanisms rotate the rotating assemblies, which engage the belts positively or frictionally, to drive the belts in the directions shown.

The apparatus also includes a diverter element 44 that extends across at least a portion of the width of the first and second conveyors. The diverter element is attached to a carrier, in this example, an intermediate belt 46 disposed in the space 48 between the infeed and outfeed belts. The intermediate belt is preferably a modular plastic conveyor belt, but could as well be a metal chain or a fabric belt. Only a portion of the intermediate belt is shown in FIGS. 1 and

2 in order to show other features of the apparatus, but the intermediate belt is co-extensive with the infeed and outfeed belts in that it is also an endless belt wrapped around the first and second rotating assemblies 32, 33. The intermediate belt, positioned between the infeed and outfeed belts, abuts each of them with only a minimal gap. (Siderails to confine articles on the conveyor are not shown to simplify illustration.)

The diverter element 44 captures articles 50 from a stream of articles on the infeed belt 28 and directs them across a transfer surface 52 of the intermediate belt onto the outfeed belt 30 traveling in the opposite direction. The diverter element has a guide surface 54, in this example, an arcuate surface, upstanding from a base 56. The base is attached to a section of the intermediate belt so that the diverter element moves with the intermediate belt. Arms 60, 61 of the diverter element extend across at least a portion of the conveying surfaces of the infeed and outfeed belts. In the version shown in FIGS. 1 and 2, the first arm 60 extends across most of the width of the infeed belt. In this case, all of the articles 50 in the stream are received by the diverter element. But it is also possible to have a shorter first arm that would capture some articles, but allow others to pass. The arms can ride directly on the infeed and outfeed conveyors, or they can be positioned above the conveying surfaces as long as they are not so high as to allow articles to be diverted to pass underneath in normal operation. A barrier 62 is also attached to the intermediate belt. The guide surface of the diverter element and the barrier bound the transfer surface 52 of the intermediate belt. The barrier ensures that products are diverted only across the transfer surface and not at other points along the carryway. For smooth transfer of product, the conveying surfaces of the infeed and outfeed belts are preferably coplanar with the outer surface of the intermediate belt. It would also be possible, as shown in FIG. 3, to have the conveying surfaces of the infeed 28 and outfeed 30 conveyor belts and the outer surface of the intermediate belt 46 stepped down in elevation from the infeed belt to the outfeed belt. Alternatively, the entire conveyor bed could be slanted downward, off horizontal, toward the outfeed belt to get an assist from gravity in clearing articles off the transfer surface.

In the exemplary version of the apparatus shown in FIGS. 1 and 2, the rotating assemblies 32 at the first end 24 of the conveyor bed are shown in more detail in FIG. 4. A first rotating assembly 64 and a second rotating assembly 65 are coaxially arranged and terminate in a differential drive mechanism 66. The first rotating assembly actively drives the outfeed belt 30. The assembly is stationarily mounted to the crossbeam 23. The assembly includes the drive motor 42, which, in this example, has a right-angle drive shaft 68. The first rotating assembly includes a shaft 69, which is shown segmented, on which sprockets 70 are

mounted. Bearing blocks 72 support the shafts for rotation. The numbers of bearing blocks, sprockets, and shaft segments required depend on the width of the belt and its load. In this example, four sprockets, three bearing blocks, and two shaft segments are shown for the first rotating assembly. The sprocket shaft 69 is connected to the motor drive shaft 68 by a coupling 74. The other end of the sprocket shaft is connected to a differential shaft 76 (FIG. 5) by a coupling 77. The sprockets have peripheral teeth (shown in FIG. 4 as a raised portion to simplify the drawing) that engage underside receiving structure in the outfeed belt, or chain, to drive and track it. In the case of a fabric belt, the toothed sprockets are replaced by roller pulleys that frictionally drive the belt.

While the first rotating assembly 64 at the first end of the belt forms a drive mechanism for the outfeed belt, the second rotating assembly 65 is an idler assembly for the infeed belt 28. In this version of the apparatus, the drive mechanism for the infeed conveyor is in the rotating assemblies 33 stationed at the other end 25 of the conveyor bed. Although this arrangement is preferable because both drive mechanisms are pulling the belts, the drives could be stationed at the same end or, especially for long belts, distributed along the length of the belts. The idler assembly 65 is similar to the first rotating assembly 64, but is not directly coupled to a motor. It is, however, coupled to the differential drive mechanism by a coupling 77. Thus, the differential drive mechanism is coupled to each rotating assembly, which rotates at a speed corresponding to the speed of the associated belt. Each rotating assembly, whether including a drive shaft or an idler shaft, acts as an input drive mechanism to the differential drive mechanism.

The differential drive mechanism is shown in more detail in FIG. 5. The mechanism includes four bevel gears 78 A-D. The differential shaft 76, rotating at the speed of the outfeed belt, is connected to the bevel gear 78A. The differential shaft 76', rotating at the speed of the infeed belt, is connected to the opposite bevel gear 78B. The gears mesh with gears 78C and 78D, which are coaxially and rotatably aligned as pinion gears on pinion shaft 80. Couplings 82 retain the bevel gears in place on the shafts and pinion shaft. The ends of the pinion shaft extend from a spider 83, which also provides supports for the differential shafts 76, 76'. The differential mechanism fits in a hollow 84 formed in the center of two mating central housing halves 86. The ends of the pinion shaft fit in cavities 88 formed radially in the housing halves. Metal plates 89 serve as thrust bearings. Dowels 90 register the two housing halves, which are held together conventionally by bolts or screws through holes 92. A toothed sprocket wheel 94 is attached to each housing half. The peripheral teeth of the sprocket wheel engage the intermediate belt 46 to drive it.

The geared differential works conventionally in that relative motion of the shaft output bevel gears 78A and 78B causes the pinion gears 78C and 78D to rotate about the axis of the differential shafts 76, 76'. As the pinion gears rotate, the ends of the pinion shaft 80 cause the housing and the sprocket wheels to rotate. The speed of rotation depends on the relative
5 speeds of the rotation of the output shaft bevel gears. In the situation where the outfeed belt and the infeed belt are moving at the same speed in opposite directions, the outfeed output bevel gear 76 rotates in one direction at a certain speed and the infeed output bevel gear 76' rotates in the opposite direction at the same speed, which causes the pinion gear assembly to rest with its pinion shaft stationary. As one of the conveyor belts speeds up relative to the
10 other, the differential drive mechanism causes the housing and sprocket wheel assembly to rotate in the direction of the faster moving rotating assembly, but at half the difference between the speeds of each rotating assembly. Thus, in this example, the speed s of the intermediate belt is given by $s = \frac{1}{2} (s_1 - s_2)$, where s_1 is the speed of the faster-moving belt and s_2 is the speed of the slower-moving belt. Of course, the gearing ratios can be altered by the use of gear
15 reducers or other conventional techniques to derive other speed relationships that may be generically defined by s is proportional to $as_1 - bs_2$, where a and b are parameters set by the effective gear ratios, for example. This would allow the intermediate belt to be driven at a speed that is influenced relatively more by one of the conveyor belts than the other in special applications.

20 The second rotating assemblies 33 stationed at the second end 25 of the conveyor bed are similar to those at the first with the following difference shown in FIG. 6. First, the drive motor 42 for the infeed belt and its drive elements 96 are mounted on the crossbeam 23 at this end. Second, idler elements 98 form the rotating assembly for the infeed belt. Third, the rotating elements, although mounted coaxially, do not drive a differential mechanism at this
25 end. Instead, an idler sprocket 100 on an idler shaft 102 supported by flanking bearing blocks 72 is provided to support and track the endless intermediate belt 46 at this end of the conveyor bed.

Another version of an apparatus for diverting articles is shown in FIG. 7. In this version, the infeed belt 28 and the outfeed belt 30 abut each other. The diverter element 44' is
30 carried by a carrier belt 104 next to, in this example, the infeed belt. The diverter element extends through a gap between stationary side rails 106 across the conveying surfaces of the infeed and outfeed conveyor belts to direct articles from the first belt directly to the oppositely running second belt without having to traverse an intermediate belt – a distinct advantage. The carrier belt is driven differentially by the drive mechanisms of each of the conveyor belts.

Because, in this version, the carrier belt is not adjacent to the outfeed belt, the drive mechanism of the outfeed belt would have to be connected with the differential drive mechanism of the carrier belt through an intervening jackshaft or some other conventional linkage.

A slightly modified alternative to the apparatus in FIG. 7 is illustrated in FIG. 8. In this version, a diverter element 44" is carried at each end by a carrier belt 104. This version provides better support for the diverter element. The two flanking carrier belts are each driven differentially in unison.

The versions described thus far are especially adaptable to be retrofitted into existing bi-di tables, which typically provide for sprockets, shafts, and motors at each end and include belt-supporting structure along the carryway and returnway. Essentially, the bi-di belt and its drive and idler sprocket assemblies are replaced by the belts and rotating assemblies of the invention. The remainder of the bi-di conveyor bed remains largely unchanged. As already mentioned, the versions described lend themselves to easy expansion through widening or lengthening of the belts.

To this point, the carrier for the diverter element has been described as a belt (an intermediate belt or a carrier belt) differentially driven by a differential drive mechanism connected to infeed and outfeed belt drive mechanisms. In another version of the apparatus, shown in FIGS. 9 and 10, the diverter element 44 is carried on a carrier 106, which is the linearly variable nut of a lead screw 108. The lead screw is connected at one end to a stationary differential gear 110 and at the other to a fixed bearing block 112 at the opposite end of the carrier run. The speed and direction of rotation of the lead screw depend on the relative speeds of the infeed 28 and outfeed 30 belts. As the lead screw rotates, the carrier is driven linearly along its run. To minimize the space between the two conveyor belts, the diverter element is connected to the carrier by a thin, but sturdy, fin 114 that is attached to each by welding, for instance.

All of these diverter carrier arrangements can be used to divert articles from one conveyor to another and are useful, for example, in a first-in, first-out article accumulator, such as the one shown in FIG. 11. The accumulator system 116 effectively adjusts the length of a conveying path depending on the upstream supply of and the downstream demand for articles. A stream of articles 50 conveyed in the direction of arrow 118 by a main conveyor line 120 is received at an upstream location 122 of the main conveyor by an infeed conveyor belt 28 driven in the direction of arrow 34. An infeed guide rail 124 guides the stream of articles onto the infeed belt. As the infeed belt transports the articles, they are captured and redirected by the diverter element 44 attached to and carried by the intermediate belt 46. The articles

transfer across the transfer surface 52 of the intermediate belt between the diverter element and the barrier 62 and onto the outfeed conveyor belt 30. The diverted articles are delivered to the main conveyor at a downstream location 128 by the outfeed conveyor belt driven toward the main conveyor. An outfeed guide rail 125 guides the stream of articles back onto the main conveyor. A transfer plate 126, such as a deadplate, is used for a smooth transfer of articles to and from the main conveyor. The main conveyor could alternatively use other transfer techniques, such as transfer belt edges attached to and moving with the main conveyor belt. The infeed and outfeed belts need not be perpendicular to the conveyor, but could, for example, run parallel to a main conveyor line that includes upstream and downstream segments on opposite sides of the infeed/outfeed belt unit. The accumulator allows the transport of articles along the main conveyor line to be adjusted for mismatches between the upstream supply of and the downstream demand for articles. The accumulator does this by controlling the effective length of the conveying path. As the supply of articles increases, the infeed belt is driven at a higher speed than the outfeed belt. The differentially driven carrier belt moves away from the main conveyor. With the diverter element farther away from the main conveyor, more articles can accumulate on the lengthened conveying path. As the demand for articles downstream increases to a level outstripping the supply of articles, the outfeed conveyor is driven faster than the infeed conveyor. When this occurs, the carrier belt and its diverter element move toward the main conveyor and shorten the effective conveying path and the proportional accumulation area. For protection, limit switches (not shown) at each end of the infeed/outfeed conveyor bed cause the motors to stop when the diverter element gets to either end of its permissible range along the conveyor bed.

The supply of and demand for articles can be sensed by conventional article sensors 130 positioned at, for example, upstream and downstream locations on the main conveyor. Their signals 132, 133 are electrically directed to a controller 134, which further sends control signals 136, 137 to the variable-speed drive motors 42 to adjust the speeds of the infeed and outfeed belts. Although the differential drive mechanisms described to this point are mechanically-coupled differentials, it would alternatively be possible to have an independent drive for the carrier with the differential speed calculated by the controller and an appropriate speed control signal sent to the carrier's differential drive mechanism. With such an arrangement, the controller easily tailors the speed relationship (e.g., $s = as_1 - bs_2$) to the needs of the system at any time. An operator adjusts the values of the a and b parameters through the controller.

Thus, the invention has been described with respect to various versions by way of example. Even so, those skilled in the art will appreciate that other versions are possible without materially departing from the novel teachings and advantages of the invention. For example, the carrier could ride on a rack and pinion arrangement, cable and drum arrangement,
5 or a drag chain and rail arrangement. As another example, multiple intermediate drive mechanisms, including multiple differentials, could be distributed along the conveyor, especially for long belts. The belts could be radius, sideflexing belts arranged along a curved path. The differential could be realized with spur gears or other gear types. As the examples suggest, these and other modifications are intended to be included within the scope of the
10 invention as defined in the following claims.

What is claimed is:

Claims

1. Apparatus for diverting a stream of articles, comprising:
a first conveyor belt conveying a stream of articles along a conveying surface;
a second conveyor belt disposed parallel to the first conveyor belt and having a
5 conveying surface;
an endless intermediate belt disposed parallel to and abutting the first and second
conveyor belts and having a transfer surface;
a first drive mechanism for driving the first conveyor belt at a first speed in a first
direction;
10 a second drive mechanism for driving the second conveyor belt at a second speed in a
second direction opposite to the first direction;
a differential drive mechanism for driving the intermediate belt at a speed and in a
direction that depend on the first speed and the second speed; and
a diverter element affixed to the intermediate belt and extending across at least a
15 portion of the conveying surfaces of the first and second conveyor belts to
divert articles in the stream on the first conveyor belt across the transfer surface
of the intermediate belt onto the second conveyor belt.
2. Apparatus as in claim 1, wherein the differential drive mechanism drives the
intermediate belt at a speed equal to half the difference between the first speed and the
20 second speed.
3. Apparatus as in claim 1, wherein the differential drive mechanism drives the
intermediate belt at a speed proportional to $as_1 - bs_2$, where s_1 is the first speed, s_2 is the
second speed, and a and b are adjustable parameters.
4. Apparatus as in claim 1, wherein the conveying surfaces of the first and second
25 conveyor belts and the transfer surface of the intermediate belt are coplanar.
5. Apparatus as in claim 1, wherein the conveying surface of the first conveyor belt is
elevated above the transfer surface of the intermediate belt, and the transfer surface of
the intermediate belt is elevated above the conveying surface of the second conveyor
belt.
- 30 6. Apparatus as in claim 1, wherein the conveying surfaces of the first and second
conveyor belts and the transfer surface of the intermediate belt are co-extensive in the
first and second directions.
7. Apparatus as in claim 1, wherein the diverter element includes a first arcuate arm
extending at least partway across the conveying surface of the first conveyor belt and a

- second arcuate arm extending at least partway across the conveying surface of the second conveyor belt.
8. Apparatus as in claim 1, further comprising a controller for directly controlling the first drive mechanism and the second drive mechanism.
- 5 9. Apparatus as in claim 8, wherein the controller directly controls the differential drive mechanism.
10. Apparatus as in claim 1, wherein the differential drive mechanism includes a differential gear arrangement connected to the first drive mechanism and to the second drive mechanism.
- 10 11. Apparatus for diverting a stream of articles, comprising:
a framework;
a first conveyor belt supported in the framework and conveying a stream of articles along a conveying surface;
a second conveyor belt supported in the framework parallel to the first conveyor belt
15 and having a conveying surface;
a carrier arranged to travel along a path parallel to the first and second conveyor belts;
a first drive mechanism for driving the first conveyor belt at a first speed in a first direction;
a second drive mechanism for driving the second conveyor belt at a second speed in a
20 second direction opposite to the first direction;
a differential drive mechanism stationarily connected to the framework and drivingly coupled to the carrier for driving the carrier along the path at a speed and in a direction that depend on the first speed and the second speed; and
a diverter element affixed to the carrier and extending across at least a portion of the
25 conveying surfaces of the first and second conveyor belts to divert articles in the stream on the first conveyor belt to the second conveyor belt.
12. Apparatus as in claim 11, wherein the carrier comprises an endless conveyor belt arranged parallel to and between the first and second conveyor belts.
13. Apparatus as in claim 11, further comprising a lead screw, extending from the
30 differential drive mechanism parallel to the first and second conveyor belts, along which the carrier rides.
14. Apparatus as in claim 11, wherein the differential drive mechanism drives the carrier at a speed equal to half the difference between the first speed and the second speed.

15. Apparatus as in claim 11, wherein the differential drive mechanism drives the carrier at a speed proportional to $as_1 - bs_2$, where s_1 is the first speed, s_2 is the second speed, and a and b are adjustable parameters.
16. Apparatus as in claim 11, further comprising a controller for directly controlling the first drive mechanism and the second drive mechanism.
17. Apparatus as in claim 16, wherein the controller directly controls the differential drive mechanism.
18. Apparatus as in claim 11, wherein the differential drive mechanism includes a differential gear arrangement connected to the first drive mechanism and to the second drive mechanism.
19. Apparatus for diverting a stream of articles, comprising:
a first conveyor belt conveying a stream of articles along a conveying surface;
a second conveyor belt disposed parallel to the first conveyor belt and having a conveying surface;
a carrier arranged to travel along a path parallel to the first and second conveyor belts;
a first drive mechanism for driving the first conveyor belt at a first speed in a first direction;
a second drive mechanism for driving the second conveyor belt at a second speed in a second direction opposite to the first direction;
a differential drive mechanism drivingly connected to the first and second drive mechanisms and drivingly coupled to the carrier for driving the carrier along the path at a speed and in a direction that depend on the first speed and the second speed; and
a diverter element affixed to the carrier and extending across at least a portion of the conveying surfaces of the first and second conveyor belts to divert articles in the stream on the first conveyor belt to the second conveyor belt.
20. Apparatus as in claim 19, wherein the carrier comprises an endless conveyor belt arranged parallel to and between the first and second conveyor belts.
21. Apparatus as in claim 19, further comprising a lead screw, extending from the differential drive mechanism parallel to the first and second conveyor belts, along which the carrier rides.
22. Apparatus as in claim 19, wherein the differential drive mechanism drives the carrier at a speed equal to half the difference between the first speed and the second speed.

23. Apparatus as in claim 19, wherein the differential drive mechanism drives the carrier at a speed proportional to $as_1 - bs_2$, where s_1 is the first speed, s_2 is the second speed, and a and b are adjustable parameters.
24. Apparatus as in claim 19, further comprising a controller for directly controlling the first drive mechanism and the second drive mechanism.
25. Apparatus as in claim 24, wherein the controller directly controls the differential drive mechanism.
26. Apparatus as in claim 19, wherein the differential drive mechanism is disposed in a stationary position.
27. Apparatus for diverting a stream of articles, comprising:
a first conveyor belt conveying a stream of articles along a conveying surface in a first direction;
a second conveyor belt disposed parallel to the first conveyor belt and having a conveying surface;
a carrier belt disposed parallel to the first and second conveyor belts;
a diverter element affixed to the carrier belt and extending across at least a portion of the conveying surfaces of the first and second conveyor belts to divert articles from the stream of articles on the first conveyor belt traveling in the first direction onto the second conveyor belt traveling in a second direction opposite the first direction;
a first rotating assembly rotating at a first speed about an axis of rotation and about which the first conveyor belt articulates in engagement;
a second rotating assembly rotating at a second speed about the axis of rotation and about which the second conveyor belt articulates in engagement; and
a differential drive mechanism rotatable about the axis of rotation and drivingly engaged with the carrier belt and rotationally coupled to the first rotating assembly and to the second rotating assembly to drive the carrier belt at a speed and in a direction that depend on the first speed and the second speed.
28. Apparatus as in claim 27, wherein the differential drive mechanism drives the carrier belt at a speed equal to half the difference between the first speed and the second speed.
29. Apparatus as in claim 27, wherein the differential drive mechanism drives the carrier belt at a speed proportional to $as_1 - bs_2$, where s_1 is the first speed, s_2 is the second speed, and a and b are adjustable parameters.

30. Apparatus as in claim 27, wherein the carrier belt is disposed between and abuts the first and second conveyor belts and the carrier belt includes a transfer surface.
31. Apparatus as in claim 30, wherein the conveying surfaces of the first and second conveyor belts and the transfer surface of the carrier belt are coplanar.
- 5 32. Apparatus as in claim 30, wherein the conveying surfaces of the first and second conveyor belts and the transfer surface of the carrier belt are co-extensive in the first and second directions.
33. Apparatus as in claim 27, wherein the first and the second conveyor belts abut each other and wherein the carrier belt is disposed adjacent to the first conveyor belt
10 opposite the second conveyor belt.
34. Apparatus as in claim 33, further comprising a second carrier belt disposed adjacent to the second conveyor belt so that the carrier belts flank the abutting first and second conveyor belts, and wherein the diverter element is attached to both carrier belts and extends across the conveying surfaces of the first and second conveyor belts.
- 15 35. Apparatus as in claim 27, wherein the diverter element includes a first arcuate arm extending at least partway across the conveying surface of the first conveyor belt and a second arcuate arm extending at least partway across the conveying surface of the second conveyor belt.
36. Apparatus as in claim 27, further comprising a controller for directly controlling the
20 first drive mechanism and the second drive mechanism.
37. Apparatus as in claim 36, wherein the controller directly controls the differential drive mechanism.
38. Apparatus as in claim 27, wherein the differential drive mechanism includes a differential gear arrangement connected to the first drive mechanism and to the second
25 drive mechanism.
39. Apparatus as in claim 27, wherein the first rotating assembly includes a first sprocket set mounted on a first shaft and engaged with the first conveyor belt and wherein the second rotating assembly includes a second sprocket set mounted on a second shaft and engaged with the second conveyor belt and wherein the differential drive mechanism
30 includes a first gear at an end of the first shaft, a second gear at the end of the second shaft, a differential gear meshed with the first and second gears, and a third sprocket set coupled to the differential gear for rotation and engaged with the carrier belt.
40. Apparatus for diverting the flow of a stream of articles, comprising:
an infeed conveyor including a first belt;

- an infeed drive mechanism for driving the first belt in a first direction;
a main conveyor supplying a stream of articles to the infeed conveyor at an upstream location of the main conveyor;
an outfeed conveyor including a second belt arranged parallel to the first belt;
5 an outfeed drive mechanism for driving the second belt in a second direction to supply the stream of articles to the main conveyor at a downstream location of the main conveyor;
a carrier arranged to travel along a path parallel to the first and second belts;
a diverter element attached to the carrier and extending at least partly across the first
10 belt and the second belt to divert articles from the stream of articles on the infeed conveyor onto the outfeed conveyor; and
a carrier drive mechanism engaging the infeed drive mechanism and the outfeed drive mechanism for driving the carrier along the path and the diverter element along the belts at a speed and in a direction that depends on the relative speeds of the
15 first belt and the second belt.
41. Apparatus as in claim 40, wherein the carrier drive mechanism drives the carrier at a speed proportional to $as_1 - bs_2$, where s_1 is the speed of the first belt, s_2 is the speed of the second belt, and a and b are adjustable parameters.
42. Apparatus as in claim 40, wherein the carrier is disposed to travel between the first belt
20 and the second belt.
43. Apparatus as in claim 40, wherein the carrier drive mechanism is stationary.
44. An accumulator for accumulating articles transported along a main conveyor, comprising:
a first conveyor belt receiving articles from the main conveyor at an upstream location
25 of the main conveyor onto a conveying surface, the first conveyor belt being driven in a direction away from the main conveyor at a first speed;
a second conveyor belt arranged parallel to the first conveyor belt and driven in a direction opposite to the direction of the first conveyor belt toward the main conveyor at a second speed, the second conveyor belt having a conveying
30 surface;
a carrier arranged to travel along a path parallel to the first and second conveyor belts;
a stationary differential drive mechanism for driving the carrier along the path parallel to the first and second conveyor belts at a speed depending on the difference between the first speed and the second speed; and

a diverter element attached to the carrier and extending across at least a portion of the conveying surfaces of the first and second conveyor belts to divert articles on the first conveyor belt onto the second conveyor belt, whereby articles are accumulated on the conveying surfaces of the first and second conveyor belts between the diverter element and the main conveyor before being supplied by the second conveyor belt to the main conveyor at a downstream location of the main conveyor.

45. An accumulator as in claim 44, wherein the differential drive mechanism drives the carrier along the path at a speed proportional to $as_1 - bs_2$, where s_1 is the first speed, s_2 is the second speed, and a and b are adjustable parameters.
46. An accumulator as in claim 44, wherein the carrier comprises a third belt.
47. An accumulator as in claim 46, wherein the third belt is disposed between and abutting the first and second conveyor belts.
48. An accumulator as in claim 46, further comprising a first rotating assembly including a first sprocket set mounted on a first shaft and engaged with the first conveyor belt and a second rotating assembly including a second sprocket set mounted on a second shaft and engaged with the second conveyor belt and wherein the differential drive mechanism includes a first gear at an end of the first shaft, a second gear at the end of the second shaft, a differential gear meshed with the first and second gears, and a third sprocket set coupled to the differential gear for rotation and engaged with the third belt.

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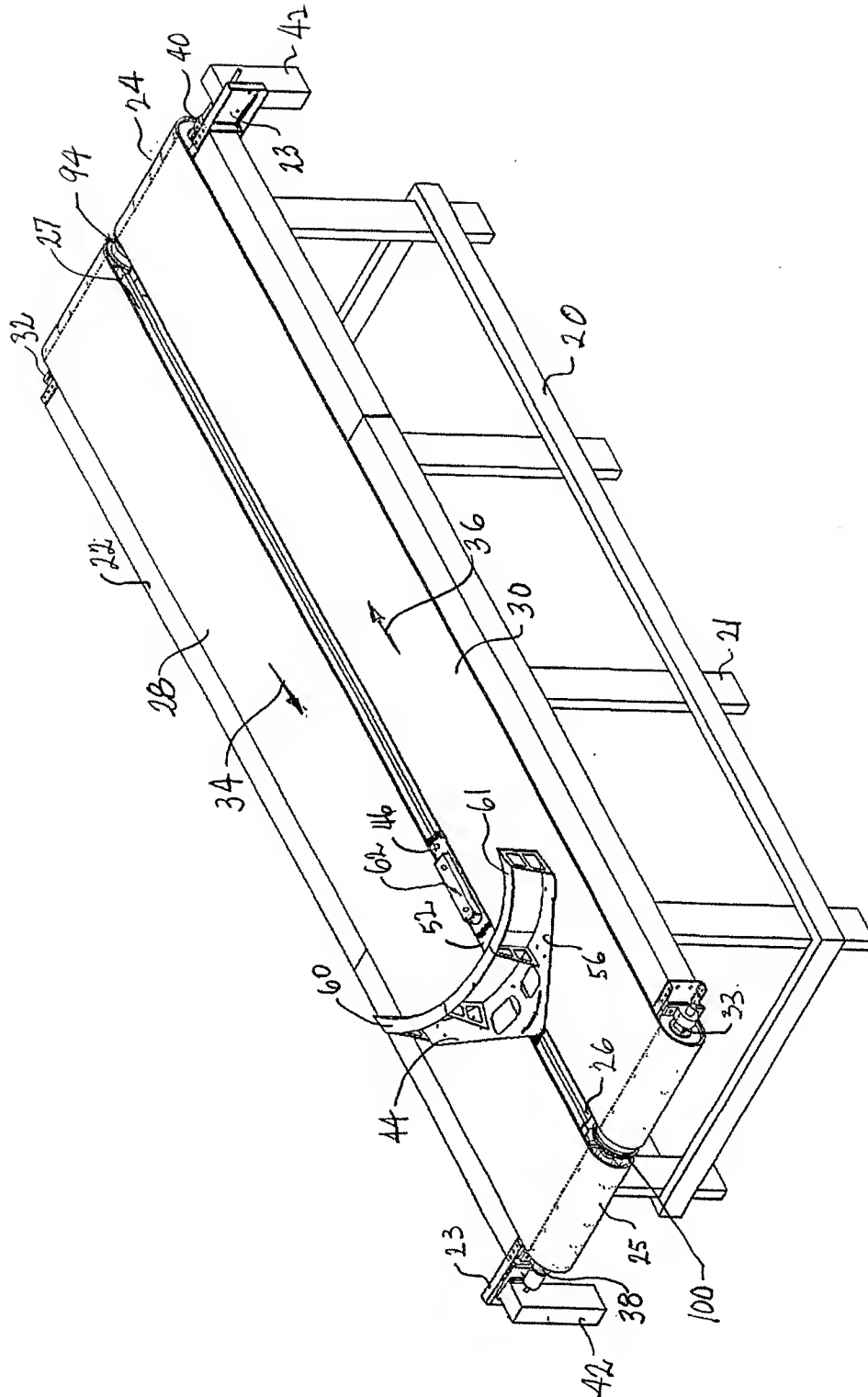


FIG. 1

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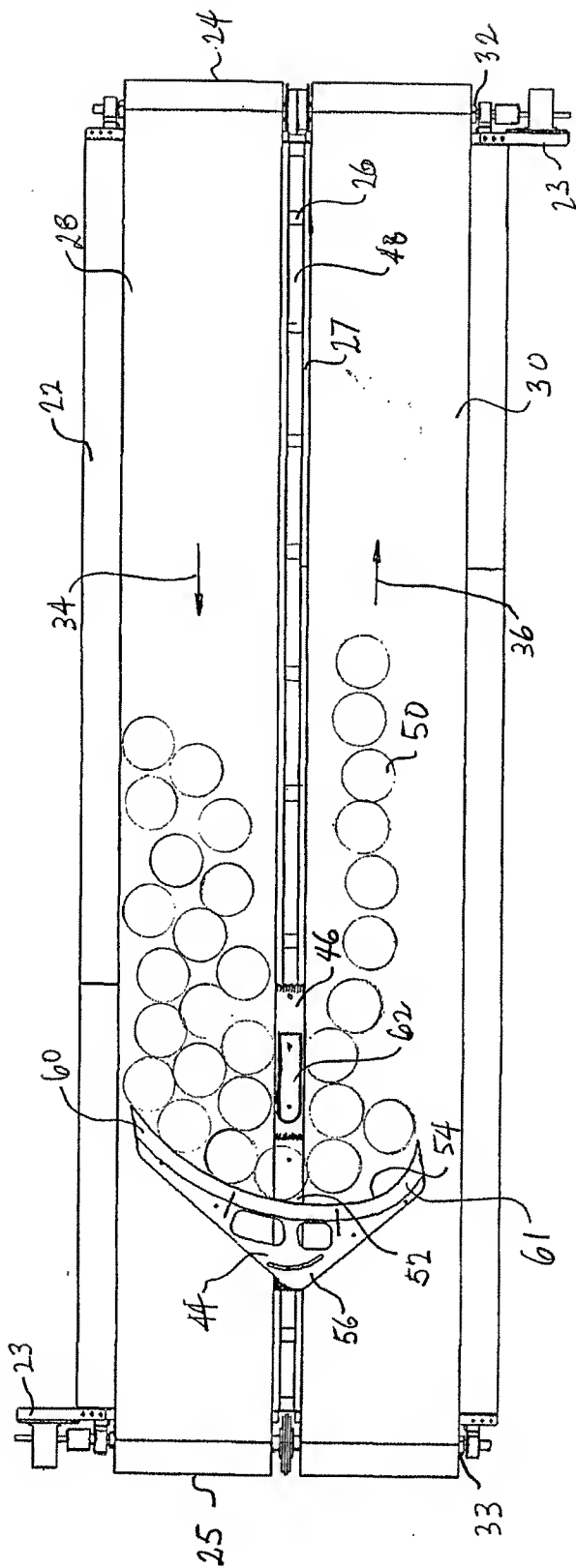


FIG. 2

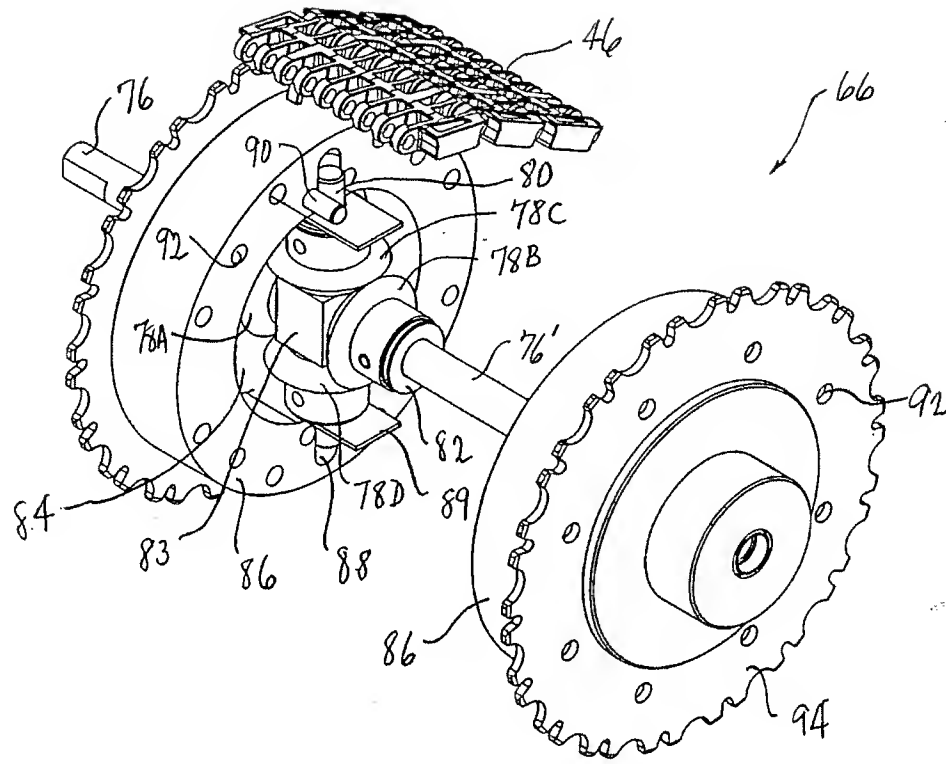


FIG. 5

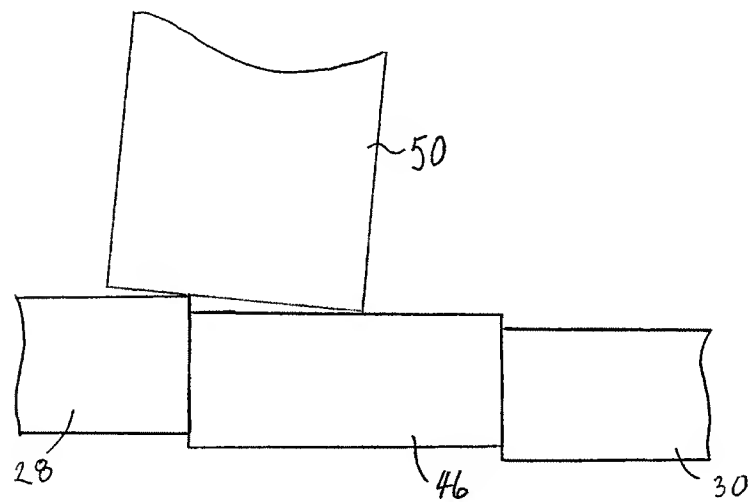
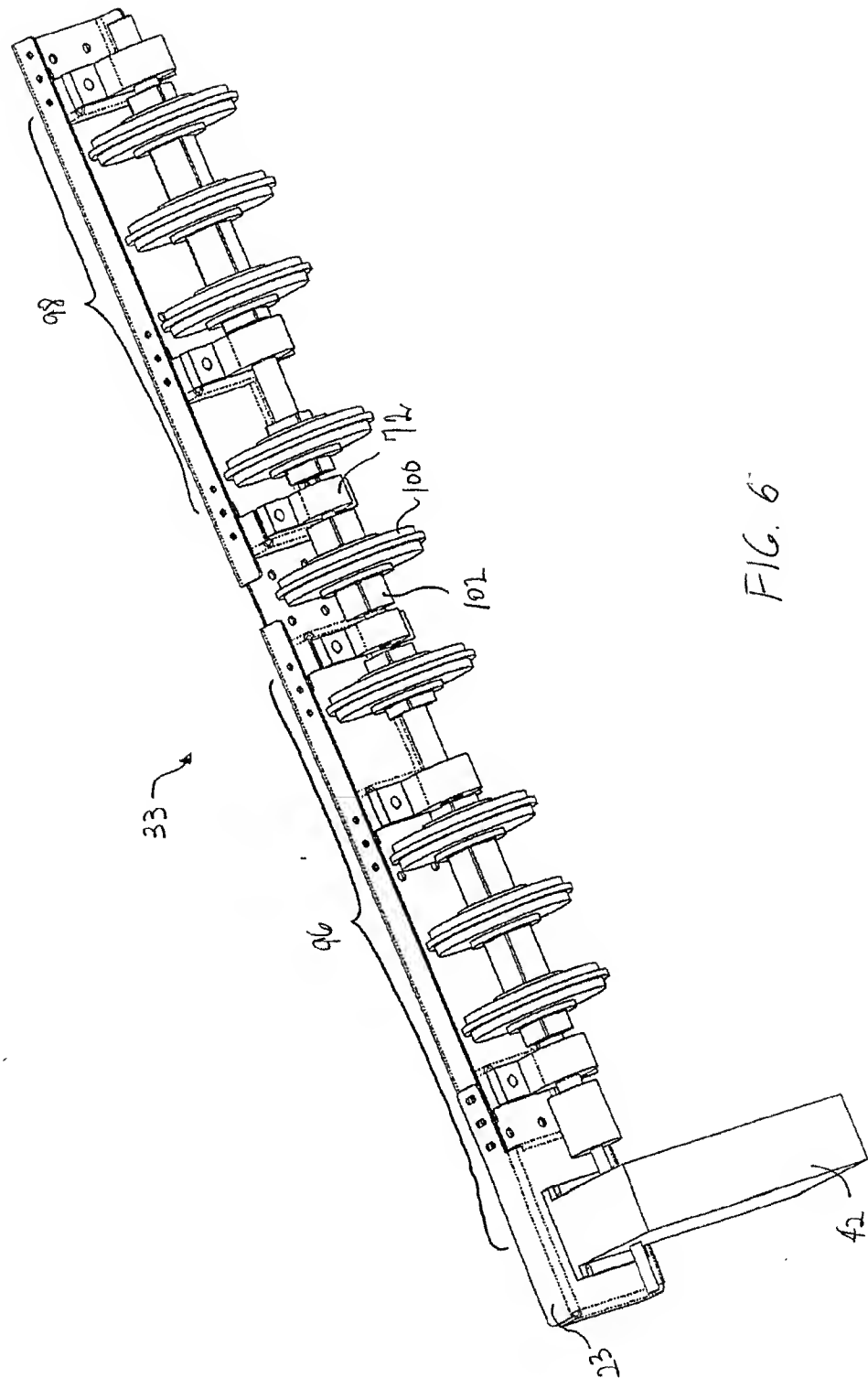


FIG. 3

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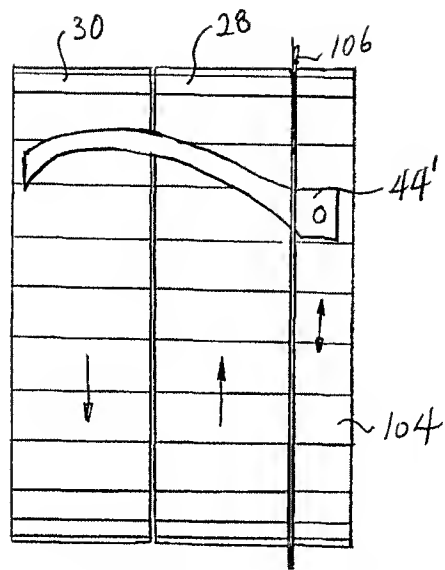


FIG. 7

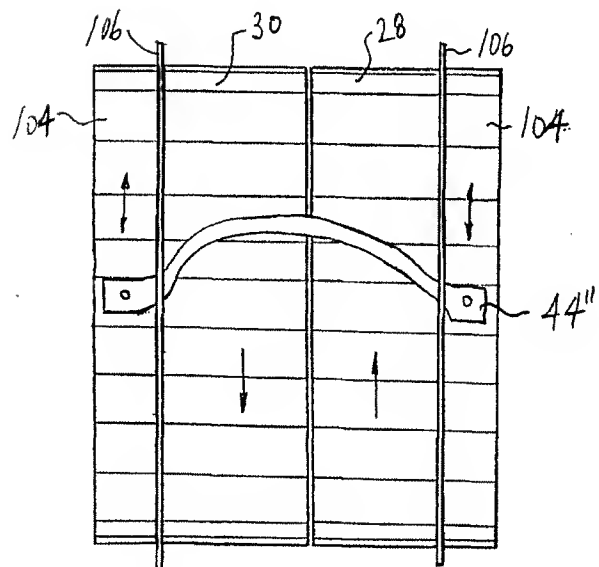


FIG. 8

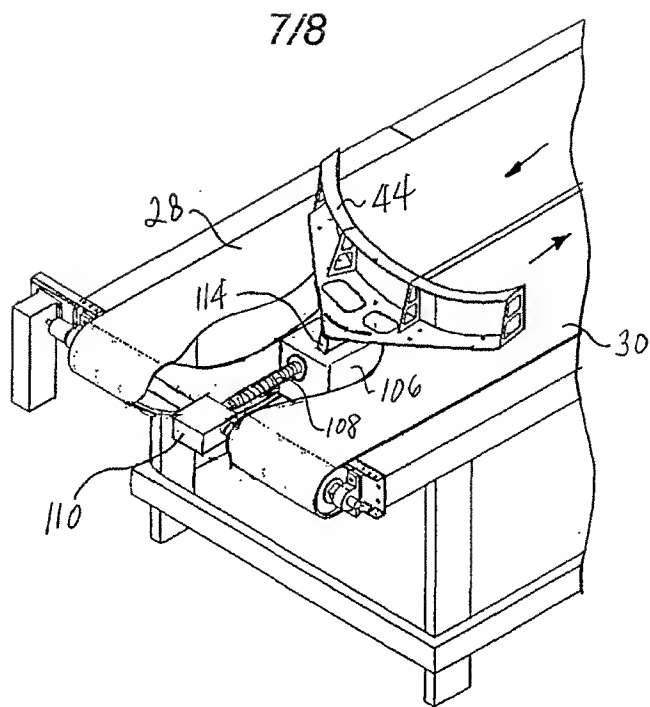


FIG. 9

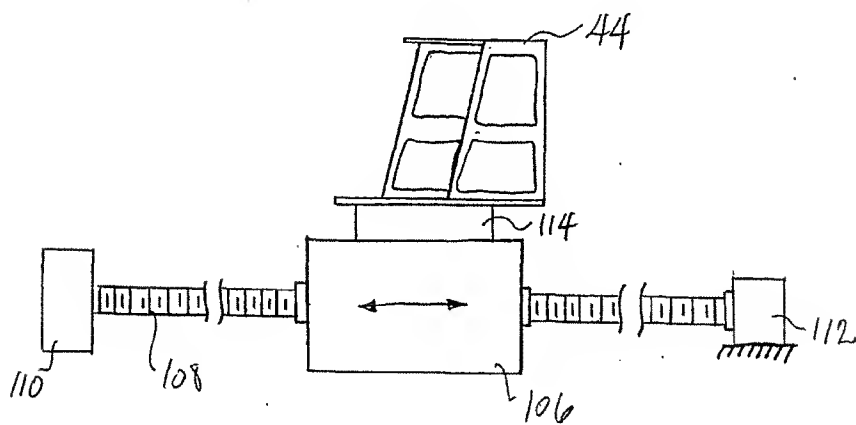
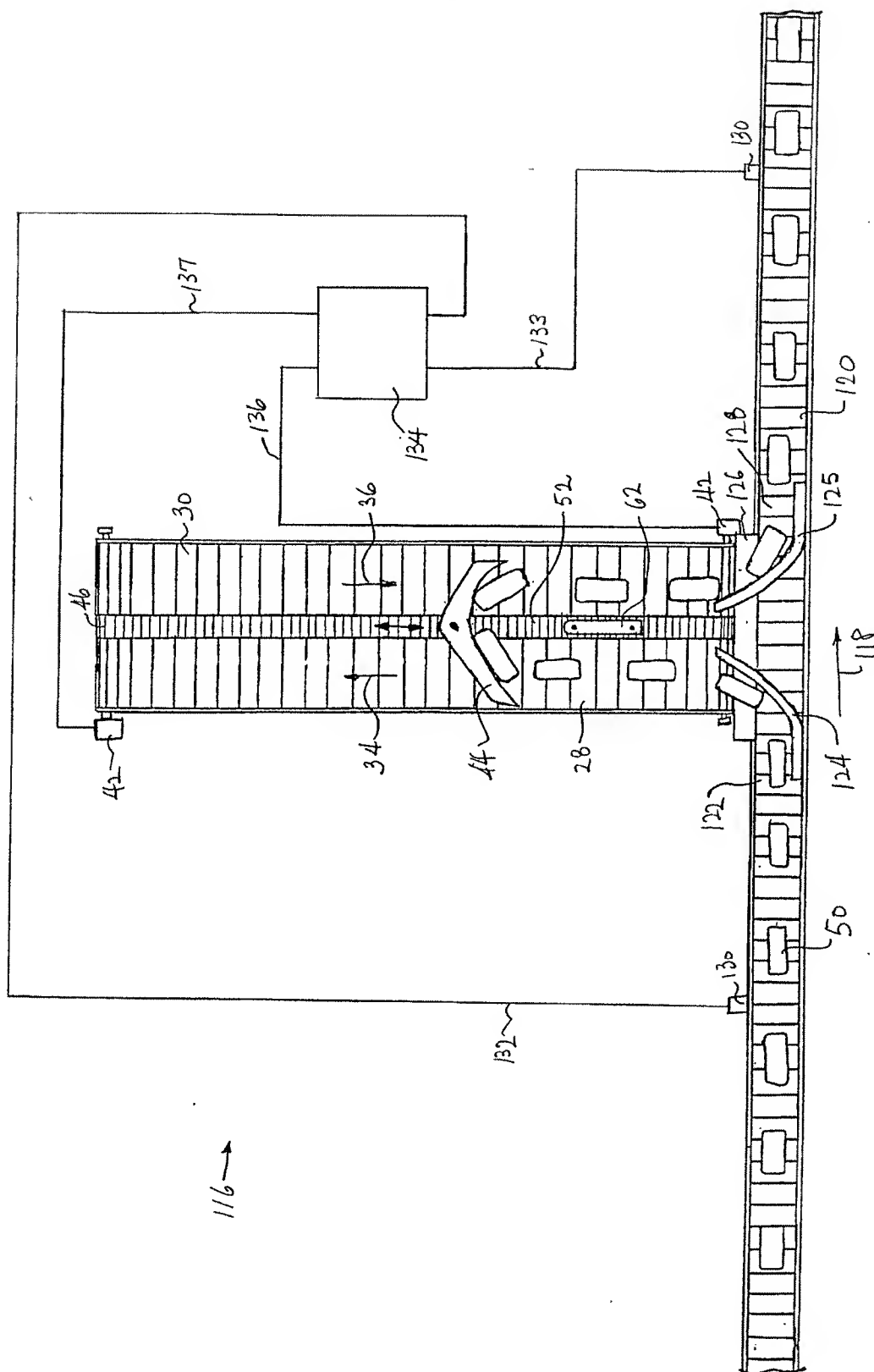


FIG. 10



F16.11

INTERNATIONAL SEARCH REPORT

International Application No
PCT/US 02/07473

A. CLASSIFICATION OF SUBJECT MATTER
IPC 7 B65G47/51

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

IPC 7 B65G

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	US 6 182 812 B1 (HARTNESS THOMAS P ET AL) 6 February 2001 (2001-02-06) the whole document ---	1-48
A	US 6 152 291 A (DUTERTE OLIVIER P ET AL) 28 November 2000 (2000-11-28) cited in the application column 8, line 13 - line 30; figure 16 ---	1-48
A	FR 2 583 394 A (MERLIN GERIN) 19 December 1986 (1986-12-19) -----	

☐ Further documents are listed in the continuation of box C.

☒ Patent family members are listed in annex.

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Date of the actual completion of the international search

9 July 2002

Date of mailing of the international search report

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INTERNATIONAL SEARCH REPORT

Information on patent family members

International Application No

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